

Research Article

An Exact Analysis of Heat and Mass Transfer Past a Vertical Plate with Newtonian Heating

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An exact analysis of heat and mass transfer past an oscillating vertical plate with Newtonian heating is presented. Equations are modelled and solved for velocity, temperature, and concentration using Laplace transforms. The obtained solutions satisfy governing equations and conditions. Expressions of skin friction, Nusselt number, and Sherwood number are obtained and presented in tabular forms. The results show that increasing the Newtonian heating parameter leads to increase velocity and temperature distributions whereas skin friction decreases and rate of heat transfer increases.

1. Introduction

Generally, the problems of free convection flows are usually modeled under the assumptions of constant surface temperature, ramped wall temperature, or constant surface heat flux [1–7]. However, in many practical situations where the heat transfer from the surface is taken to be proportional to the local surface temperature, the above assumptions fail to work. Such types of flows are termed as conjugate convective flows, and the proportionally condition of the heat transfer to the local surface temperature is termed as Newtonian heating. This work was pioneered by Merkin [8] for the free convection boundary layer flow over a vertical flat plate immersed in a viscous fluid. However, due to numerous practical applications in many important engineering devices, several other researchers are getting interested to consider the Newtonian heating condition in their problems. Few of these applications are found in heat exchanger, heat management in electrical appliances (such as computer power supplies or substation transformer), and engine cooling (such as thin fins in car radiator). Moreover, the flow over an oscillating plate with Newtonian heating also occurs in the conjugate heat transfer around fins where the conduction within the fin and the convection in the fluid surrounding it must be simultaneously analyzed in order to obtain the vital design information and in also convection

flows set up when the bounding surface absorb heat by solar radiation. The literature survey shows that much attention to the problems of free convection flow with Newtonian heating is given by numerical solvers, as we can see [9–16] and the references therein. However, the exact solutions of these problems are very few [17–21]. Exact solutions on the other hand can provide an important check for numerical methods that are used to study such flows in more complex domains.

Furthermore, the free convection flows together with heat and mass transfer are of great importance in geophysics, aeronautics, and engineering. In several process such as drying, evaporation of water at body surface, energy transfer in a wet cooling tower, and flow in a desert cooler, heat and mass transfer occurs simultaneously. In view of such applications, several authors investigated free convection flows with simultaneous heat and mass transfer phenomenon [22–25].

To the best of authors' knowledge, so far, no study has been reported in the literature which investigates the unsteady free convection flow of an incompressible viscous fluid past an oscillating vertical plate with Newtonian heating and constant mass diffusion. The present study is an attempt in this direction to fill this space. In this study, the equations of the problem are first formulated and transformed into their dimensionless forms where the Laplace transform method is applied to find the exact solutions for velocity, temperature

3. Method of Solution

The Laplace transform method solves differential equations and corresponding initial and boundary value problems. The Laplace transform has the advantage that it solves problems directly, initial value problems without determining first a general solution and nonhomogeneous differential equations without solving first the corresponding homogeneous equations. In order to obtain the exact solution of the present problem, we will use the Laplace transform technique. Applying the Laplace transform with respect to time t to the system of (9), we get

$$\begin{aligned} q\bar{u}(y, q) - u(y, 0) &= \frac{d^2\bar{u}(y, q)}{dy^2} \\ &+ \text{Gr}\bar{\theta}(y, q) + \text{Gm}\bar{C}(y, q), \\ \text{Pr}[q\bar{\theta}(y, q) - \theta(y, 0)] &= (1 + R)\frac{d^2\bar{\theta}(y, q)}{dy^2}, \\ \text{Sc}[q\bar{C}(y, q) - C(y, 0)] &= \frac{d^2\bar{C}(y, q)}{dy^2}. \end{aligned} \quad (14)$$

Here, $\bar{u}(y, q) = \int_0^\infty e^{-qt} u(y, t) dt$, $\bar{\theta}(y, q) = \int_0^\infty e^{-qt} \theta(y, t) dt$, and $\bar{C}(y, q) = \int_0^\infty e^{-qt} C(y, t) dt$ denote the Laplace transforms of $u(y, t)$, $\theta(y, t)$ and $C(y, t)$, respectively.

Using the initial condition (II), we get

$$\begin{aligned} \frac{d^2\bar{u}(y, q)}{dy^2} - q\bar{u}(y, q) + \text{Gr}\bar{\theta}(y, q) + \text{Gm}\bar{C}(y, q) &= 0, \\ \frac{d^2\bar{\theta}(y, q)}{dy^2} - q\left(\frac{\text{Pr}}{1 + R}\right)\bar{\theta}(y, q) &= 0, \\ \frac{d^2\bar{C}(y, q)}{dy^2} - q\text{Sc}\bar{C}(y, q) &= 0. \end{aligned} \quad (15)$$

The corresponding transformed boundary conditions are

$$\begin{aligned} \bar{u}(y, q) &= \frac{q}{q^2 + \omega^2}, \\ \frac{d\bar{\theta}(y, q)}{dy} &= -\gamma \left[\frac{1}{q} + \bar{\theta}(y, q) \right], \\ \bar{C}(y, q) &= \frac{1}{q} \quad \text{at } y = 0, \end{aligned} \quad (16)$$

$$\bar{u}(y, q) \longrightarrow 0, \quad \bar{\theta}(y, q) \longrightarrow 0, \quad \bar{C}(y, q) \longrightarrow 0$$

as $y \longrightarrow \infty$.

The solutions of (15) subject to the boundary conditions (16) are

$$\begin{aligned} \bar{u}(y, q) &= \frac{1}{2(q + i\omega)} e^{-y\sqrt{q}} + \frac{1}{2(q - i\omega)} e^{-y\sqrt{q}} \\ &+ \frac{ac}{q^2(\sqrt{q} - c)} e^{-y\sqrt{q}} \\ &- \frac{ac}{q^2(\sqrt{q} - c)} e^{-y\sqrt{q}\text{Pr}_{\text{eff}}} \\ &+ \frac{b}{q^2} e^{-y\sqrt{q}} - \frac{b}{q^2} e^{-y\sqrt{q}\text{Sc}}, \\ \bar{\theta}(y, q) &= \frac{c}{q(\sqrt{q} - c)} e^{-y\sqrt{q}\text{Pr}_{\text{eff}}}, \\ \bar{C}(y, q) &= \frac{1}{q} e^{-y\sqrt{q}\text{Sc}}, \end{aligned} \quad (17)$$

where $a = \text{Gr}/(\text{Pr}_{\text{eff}} - 1)$, $b = \text{Gm}/(\text{Sc} - 1)$, $c = \gamma/\sqrt{\text{Pr}_{\text{eff}}}$, and $\text{Pr}_{\text{eff}} = \text{Pr}/(1 + R)$ is the effective Prandtl number defined by Magyari and Pantokratoras [27]. By taking the inverse Laplace transform of (17) and use formulae from Appendix, we obtain

$$\theta(y, t) = F_4(y\sqrt{\text{Pr}_{\text{eff}}}, t, c), \quad (18)$$

$$C(y, t) = F_1(y\sqrt{\text{Sc}}, t), \quad (19)$$

$$\begin{aligned} u(y, t) &= \frac{1}{2} [F_5(y, t, -i\omega) + F_5(y, t, i\omega)] \\ &- \frac{a}{c^2} [F_4(y\sqrt{\text{Pr}_{\text{eff}}}, t, c) - F_4(y, t, c)] \\ &+ \frac{a}{c} [F_2(y\sqrt{\text{Pr}_{\text{eff}}}, t) - F_2(y, t)] \\ &+ a [F_3(y\sqrt{\text{Pr}_{\text{eff}}}, t) - F_3(y, t)] \\ &- b [F_3(y\sqrt{\text{Sc}}, t) - F_3(y, t)]. \end{aligned} \quad (20)$$

Note that the above solution is valid only for $\text{Pr}_{\text{eff}} \neq 1$ and $\text{Sc} \neq 1$. Moreover, the other solutions are

Case 1. When $\text{Sc} = 1$ and $\text{Pr}_{\text{eff}} \neq 1$,

$$\begin{aligned} u(y, t) &= \frac{1}{2} [F_5(y, t, -i\omega) + F_5(y, t, i\omega)] \\ &- \frac{a}{c^2} [F_4(y\sqrt{\text{Pr}_{\text{eff}}}, t, c) - F_4(y, t, c)] \\ &+ \frac{a}{c} [F_2(y\sqrt{\text{Pr}_{\text{eff}}}, t) - F_2(y, t)] \\ &+ a [F_3(y\sqrt{\text{Pr}_{\text{eff}}}, t) - F_3(y, t)] + \frac{\gamma\text{Gm}}{2} F_2(y, t). \end{aligned} \quad (21)$$

Case 2. When $Sc \neq 1$ and $Pr_{\text{eff}} = 1$,

$$u(y, t) = \frac{1}{2} [F_5(y, t, -i\omega) + F_5(y, t, i\omega)] - b [F_3(y, \sqrt{Sc}, t) - F_3(y, t)] + \frac{\gamma Gr}{2\gamma} [F_4(y, t, \gamma)] \quad (22)$$

Case 3. When $Sc = 1$ and $Pr_{\text{eff}} = 1$,

$$u(y, t) = \frac{1}{2} [F_5(y, t, -i\omega) + F_5(y, t, i\omega)] + \frac{\gamma Gr}{2\gamma} [F_4(y, t, \gamma)] - \frac{\gamma Gr}{2} \times [F_2(y, t)] + \frac{\gamma Gm}{2} [F_2(y, t)] \quad (23)$$

Here,

$$\begin{aligned} F_1(v, t) &= \text{erf} c \left(\frac{v}{2\sqrt{t}} \right), \\ F_2(v, t) &= 2\sqrt{\frac{t}{\pi}} e^{-v^2/4t} - v \text{erf} c \left(\frac{v}{2\sqrt{t}} \right), \\ F_3(v, t) &= \left(\frac{v^2}{2} + t \right) \text{erf} c \left(\frac{v}{2\sqrt{t}} \right) - v \sqrt{\frac{t}{\pi}} e^{-v^2/4t}, \\ F_4(v, t, \alpha) &= e^{(\alpha^2 t - \alpha v)} \text{erf} c \left(\frac{v}{2\sqrt{t}} - \alpha \sqrt{t} \right) - \text{erf} c \left(\frac{v}{2\sqrt{t}} \right), \\ F_5(v, t, \alpha) &= \frac{1}{2} e^{\alpha t} \left[e^{-v\sqrt{\alpha}} \text{erf} c \left(\frac{v}{2\sqrt{t}} - \sqrt{\alpha t} \right) + e^{v\sqrt{\alpha}} \text{erf} c \left(\frac{v}{2\sqrt{t}} + \sqrt{\alpha t} \right) \right], \end{aligned} \quad (24)$$

where $\text{erf} c(\cdot)$ is the complementary error function, v and α are dummy variables, and F_1, F_2, F_3, F_4 , and F_5 are dummy functions.

The dimensionless expression for skin friction evaluated from (20) is given by

$$\begin{aligned} \tau &= \frac{\tau'}{\rho U_0^2} = - \frac{\partial u}{\partial y} \Big|_{y=0} \\ &= \frac{1}{2} [e^{-i\omega t} \sqrt{-i\omega} \text{erf}(\sqrt{-i\omega t}) + e^{i\omega t} \sqrt{i\omega} \text{erf}(\sqrt{i\omega t})] \\ &\quad + \frac{a}{c} [1 - e^{c^2 t} (1 + \text{erf}(c\sqrt{t}))] \\ &\quad \times (\sqrt{Pr_{\text{eff}}} - 1) + \frac{1}{\sqrt{\pi t}} \\ &\quad + 2\sqrt{\frac{Pr_{\text{eff}} t}{\pi}} - 2a\sqrt{\frac{t}{\pi}} - 2b\sqrt{\frac{t}{\pi}} (\sqrt{Sc} - 1). \end{aligned} \quad (25)$$

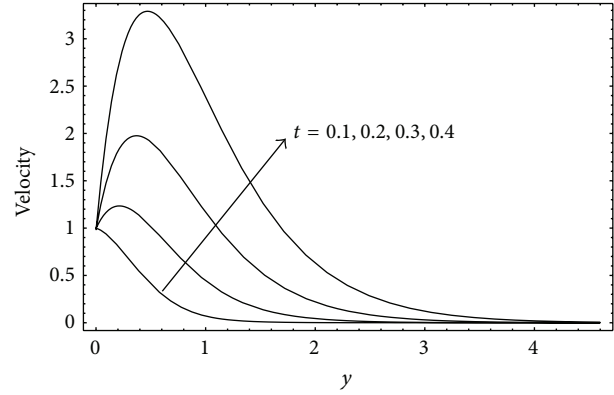


FIGURE 2: Velocity profiles for different values of t when $R = 2$, $Pr = 0.71$, $Gr = 5$, $Gm = 1$, $Sc = 0.78$, $\gamma = 1$, and $\omega = \pi/6$.

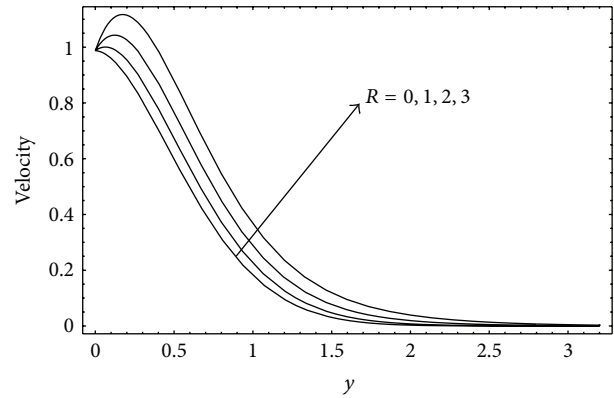


FIGURE 3: Velocity profiles for different values of R when $t = 0.2$, $Pr = 0.71$, $Gr = 2$, $Gm = 3$, $Sc = 0.94$, $\gamma = 1$, and $\omega = \pi/4$.

The dimensionless expression of Nusselt number is given by

$$\begin{aligned} Nu &= - \frac{\nu}{U_0 (T' - T_\infty)} \frac{\partial T'}{\partial y} \Big|_{y'=0} = \frac{1}{\theta(0, t)} + 1 \\ &= c \sqrt{Pr_{\text{eff}}} \left(1 + \frac{1}{e^{c^2 t} [1 + \text{erf}(c\sqrt{t})] - 1} \right). \end{aligned} \quad (26)$$

The dimensionless expression of Sherwood number is given by

$$Sh = - \frac{\partial C}{\partial y} \Big|_{y=0} = \sqrt{\frac{Sc}{\pi t}}. \quad (27)$$

4. Graphical Results and Discussion

In order to reveal some relevant physical aspects of the obtained solutions, the numerical results for velocity, temperature, and concentration are computed and shown graphically in Figures 2–16, to illustrate the influence of embedded flow parameters such as time t , radiation parameter R , Prandtl number Pr , Grashof number Gr , modified Grashof

TABLE 1: Skin friction variation.

t	R	Pr	Gr	Gm	Sc	γ	ω	τ
0.01	1	0.71	5	2	0.22	1	$\pi/2$	5.4263
0.02	1	0.71	5	2	0.22	1	$\pi/2$	3.6395
0.01	2	0.71	5	2	0.22	1	$\pi/2$	5.4049
0.01	1	1.0	5	2	0.22	1	$\pi/2$	5.4401
0.01	1	0.71	10	2	0.22	1	$\pi/2$	5.3663
0.01	1	0.71	5	4	0.22	1	$\pi/2$	5.2727
0.01	1	0.71	5	2	0.62	1	$\pi/2$	5.4537
0.01	1	0.71	5	2	0.22	2	$\pi/2$	5.3473
0.01	1	0.71	5	2	0.22	1	π	5.4208

TABLE 2: Nusselt number variation.

t	R	Pr	γ	Nu
0.2	2	0.71	1	1.3119
0.4	2	0.71	1	1.1054
0.2	4	0.71	1	1.1471
0.2	2	1.0	1	1.4659
0.2	2	0.71	2	2.0347

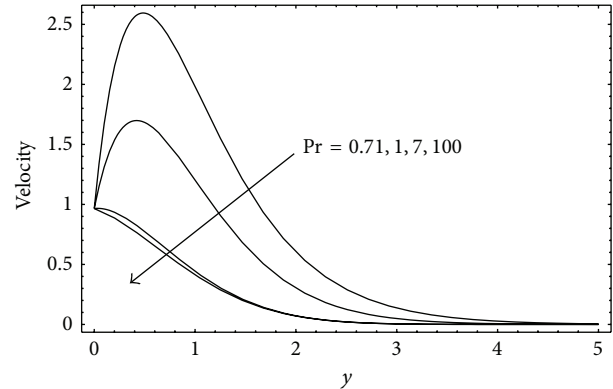
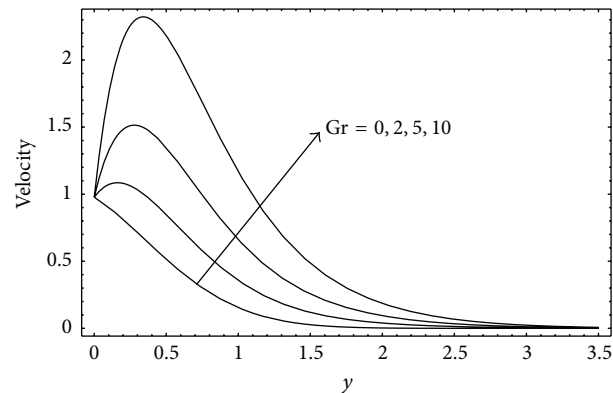
TABLE 3: Sherwood number variation.

t	Sc	Sh
0.2	0.22	0.5917
0.4	0.22	0.4184
0.2	0.62	0.9938

number Gm , Schmidt number Sc , Newtonian heating parameter γ , and phase angle ωt . The numerical values for skin friction, Nusselt number, and Sherwood number for these parameters are also presented in Tables 1–3.

The velocity profiles for different values of time t are shown in Figure 2. It is observed that the velocity increases with increasing values of time t . The velocity profiles in case of radiation and pure convection are shown in Figure 3. It is found from this figure that the radiation parameter R has an accelerating effect on velocity. Physically, it is due to the fact that an increase in the radiation parameter R for fixed values of other parameters decreases the rate of radiative heat transfer to the fluid, and consequently, the fluid velocity increases. This behavior of R is quite identical with that found in Figure 6 of Mohamed et al. [28].

The effects of Prandtl number Pr on the velocity profiles are shown in Figure 4 for $Pr = 0.71$ (air), $Pr = 1.0$ (electrolytic solution), $Pr = 7.0$ (water), and $Pr = 100$ (engine oil). It is seen from this figure that an increase in the values of Prandtl number Pr results in the decrease of velocity. In heat transfer analysis, the role of Prandtl number Pr is to control the relative thickness of the momentum and thermal boundary layers. For small value of Pr the heat diffuses very quickly compared to the velocity. This means that for liquid metals, the thickness of the thermal boundary layer is much bigger than the velocity boundary layer. The effects of Grashof number Gr and modified Grashof number Gm on velocity are shown in Figures 5 and 6. It is found that the effects of Grashof

FIGURE 4: Velocity profiles for different values of Pr when $t = 0.5$, $R = 2$, $Gr = 2$, $Gm = 1$, $Sc = 0.78$, $\gamma = 1$, and $\omega = \pi/6$.FIGURE 5: Velocity profiles for different values of Gr when $t = 0.2$, $R = 3$, $Pr = 0.71$, $Gm = 2$, $Sc = 0.78$, $\gamma = 1$, and $\omega = \pi/3$.

number Gr and modified Grashof number Gm on velocity are similar. Velocity increases with increasing values of Gr and Gm . Physically, it is possible because an increase in the values of Grashof number Gr and modified Grashof number Gm has the tendency to increase the thermal and mass buoyancy effects. This gives rise to an increase in the induced flow. Further, from these figures, it is noticed that Grashof number and modified Grashof number do not have any influence as the fluid move away from the bounding surface.

The effects of Sc on the velocity profiles are shown in Figure 7. Four different values of Schmidt number $Sc = 0.22, 0.62, 0.78$, and 0.94 are chosen. They physically correspond to hydrogen, water vapour, ammonia, and carbon dioxide, respectively. It is clear that the velocity decreases as the Schmidt number Sc increases. Further, it is clear from this figure that the velocity for hydrogen is the maximum and carbon dioxide carries the minimum velocity. Figure 8 displays the effect of Newtonian heating parameter γ on the dimensionless velocity. It is found that as the Newtonian heating parameter increases, the density of the fluid decreases, and the momentum boundary layer thickness increases and as a result, and the velocity increases within the boundary layer.

The graphical results for the phase angle ωt are shown in Figure 9. It is interesting to note that when the phase

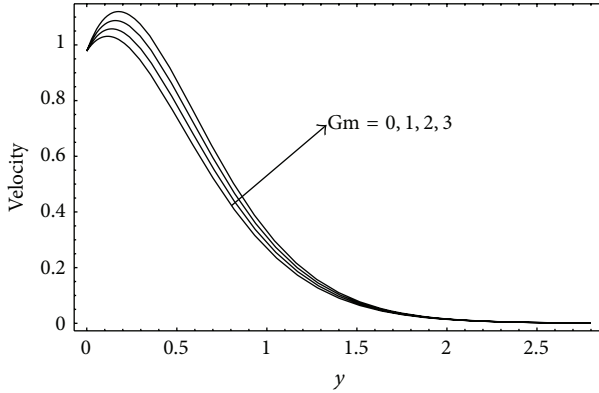


FIGURE 6: Velocity profiles for different values of Gm when $t = 0.2$, $R = 1$, $Pr = 0.71$, $Gr = 5$, $Sc = 0.78$, $\gamma = 1$, and $\omega = \pi/3$.

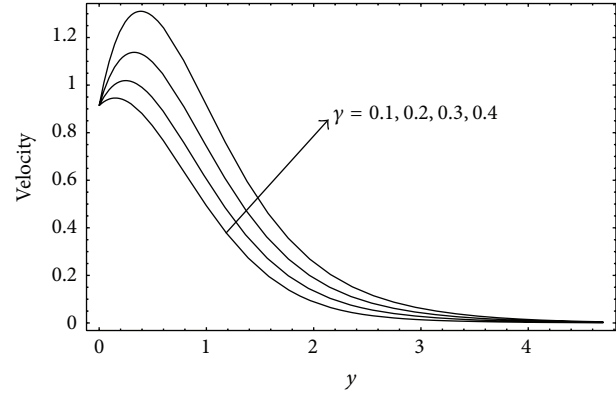


FIGURE 8: Velocity profiles for different values of γ when $t = 0.4$, $R = 3$, $Pr = 0.71$, $Gr = 5$, $Gm = 2$, $Sc = 0.78$, and $\omega = \pi/3$.

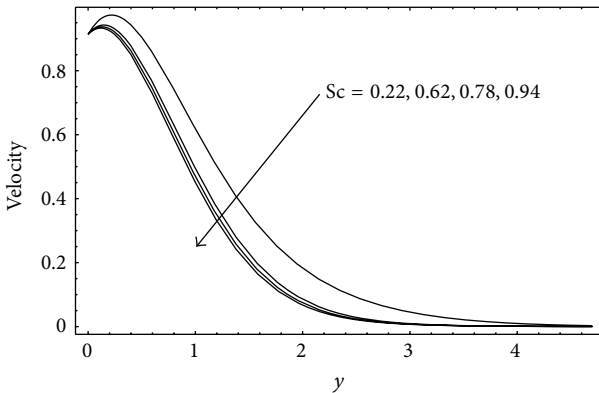


FIGURE 7: Velocity profiles for different values of Sc when $t = 0.4$, $R = 2$, $Pr = 0.71$, $Gr = 5$, $Gm = 2$, $\gamma = 0.1$, and $\omega = \pi/3$.

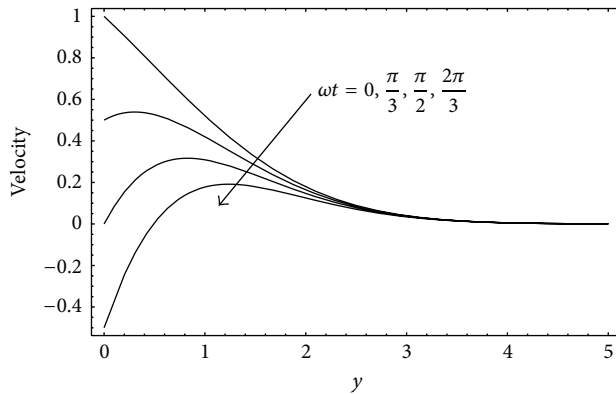


FIGURE 9: Velocity profiles for different values of ωt when $t = 1$, $R = 0.5$, $Pr = 100$, $Gr = 2$, $Gm = 0.2$, $Sc = 0.94$, and $\gamma = 0.01$.

angle ωt is zero which physically corresponds to no oscillation, then the fluid approaches to its maximum velocity of magnitude 1 meter per second, whereas for the phase angle $\omega t = \pi/2$, the velocity gains its minimum value of magnitude 0 meter per second. The oscillations near the plate are of great significance; however, these oscillations reduce for large values of the independent variable y and approach to zero as y tends to infinity. The velocity profiles are plotted in Figure 10 for different values of Sc when $\omega = 0$ (impulsive motion of the plate). It is found from this figure that the behavior of Sc on the velocity profiles quite identical with that found in Figure 6 of Narahari and Nayan [20]. Further, all these graphical results discussed above are in good agreement with the imposed boundary conditions given by (12) and (13). Hence, this ensures the accuracy of our results.

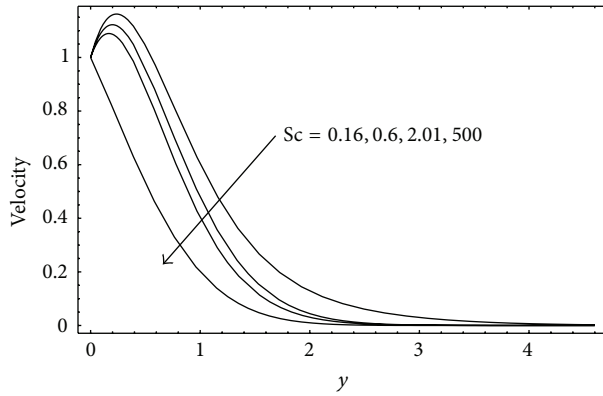
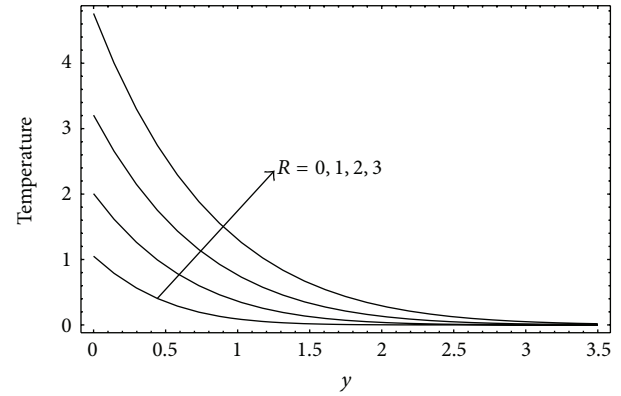
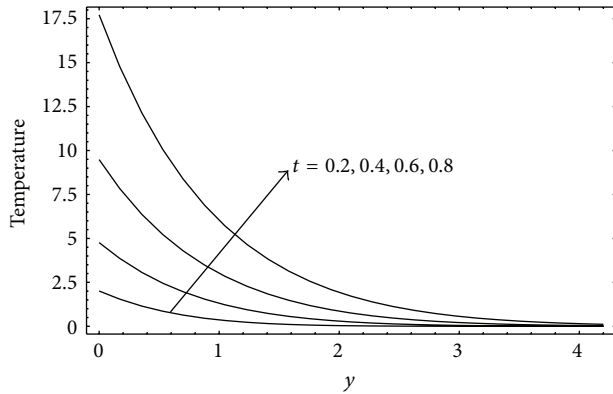
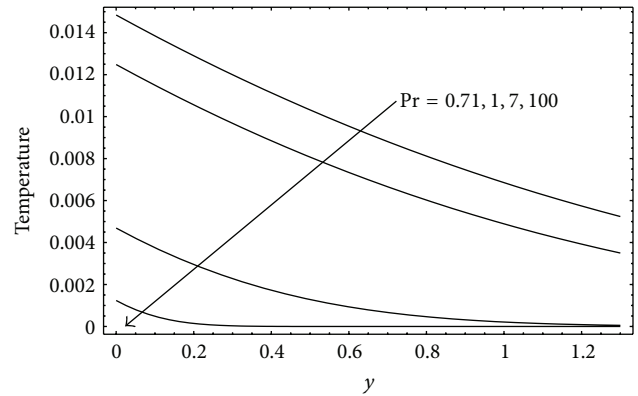
The effects of various parameters on the temperature and concentration profiles are shown in Figures 11–16. In these figures, Figure 11 exhibits the influence of dimensionless time t on the temperature. It is found that the temperature profiles increase with increasing time. From Figure 12, it is noted that an increase in the radiation parameter R leads to an increase in the temperature due to the fact that thermal boundary layer thickness of fluid increases. The influence of Prandtl number Pr on temperature profiles is shown in

Figure 13. It is found that the temperature decreases as the Prandtl number Pr increases. Physically, the increase of Pr means the decrease of thermal conductivity of fluid. From Figure 14, it is observed that an increase in the Newtonian heating parameter increases the thermal boundary layer thickness and as a result the surface temperature of the plate increases. On the other hand, it is found from Figure 15 that the influence of time t on concentration profiles is similar to the velocity and temperature profiles given in Figures 2 and 11. The effects of Schmidt number Sc on the concentration profiles are shown in Figure 16. It is seen from this figure that an increase in the value of Schmidt number makes the concentration boundary layer thin, and hence, the concentration profiles decrease.

The numerical results for skin friction, Nusselt number, and Sherwood number are shown in Tables 1, 2, and 3 for various parameters of interest. It is depicted from Table 1 that skin friction decreases with, increasing t , R , Gr , Gm , γ and ωt , while it increases as Pr and Sc are increased. Table 2 reveals that the Nusselt number increases as Pr and γ are increased and decreases when t and R are increased. From Table 3, it is observed that the Sherwood number increases with increasing Sc , while reverse effect is observed for t .

TABLE 4: Inverse Laplace transform formulae.

	$F(q) = L\{f(t)\}$	$f(t)$
1	$\frac{1}{q} e^{-y\sqrt{q}}$	$\operatorname{erfc}\left(\frac{y}{2\sqrt{t}}\right)$
2	$\frac{1}{q^2} e^{-y\sqrt{q}}$	$\left(\frac{y^2}{2} + t\right) \operatorname{erfc}\left(\frac{y}{2\sqrt{t}}\right) - y\sqrt{\frac{t}{\pi}} e^{-y^2/4t}$
3	$\frac{1}{q\sqrt{q-a}} e^{-y\sqrt{q}}$	$e^{(a^2 t - ay)} \operatorname{erfc}\left(\frac{y}{2\sqrt{t}} - a\sqrt{t}\right) - \operatorname{erfc}\left(\frac{y}{2\sqrt{t}}\right)$
4	$\frac{1}{q^2 \sqrt{q-a}} e^{-y\sqrt{q}}$	$\frac{1}{4\sqrt{\pi t}} e^{-y^2/4t} + ae^{(a^2 t - ay)} \operatorname{erfc}\left(\frac{y}{2\sqrt{t}} - a\sqrt{t}\right)$
5	$\frac{1}{q+a} e^{-y\sqrt{q}}$	$\frac{e^{at}}{2} e^{y\sqrt{a}} \operatorname{erfc}\left(\frac{y}{2\sqrt{t}} + \sqrt{at}\right) + e^{-y\sqrt{a}} \operatorname{erfc}\left(\frac{y}{2\sqrt{t}} - \sqrt{at}\right)$

FIGURE 10: Velocity profiles for different values of Sc when $t = 0.3$, $R = 0.5$, $Pr = 0.71$, $Gr = 3$, $Gm = 2$, $\gamma = 1$, and $\omega = 0$.FIGURE 12: Temperature profiles for different values of R when $t = 0.2$, $Pr = 0.71$, and $\gamma = 1$.FIGURE 11: Temperature profiles for different values of t when $R = 1$, $Pr = 0.71$, and $\gamma = 1$.FIGURE 13: Temperature profiles for different values of Pr when $t = 0.2$, $R = 5$, and $\gamma = 0.01$.

5. Conclusions

In this paper, exact solutions of unsteady free convection flow of an incompressible viscous fluid past an oscillating vertical plate with Newtonian heating and constant mass diffusion are obtained using Laplace transform technique. The results obtained show that the velocity and temperature are increased with increasing Newtonian heating parameter. Further, the effect of Newtonian heating parameter increases the Nusselt number but reduces the skin friction. However, the Nusselt number is decreased when the radiation parameter is increased. Also, the skin friction is decreased when

the radiation parameter, phase angle and Grashof number are increased. The exact solutions obtained in this study are significant not only because they are solutions of some fundamental flows, but also they serve as accuracy standards for approximate methods, whether numerical, asymptotic, or experimental.

Appendix

See Table 4.

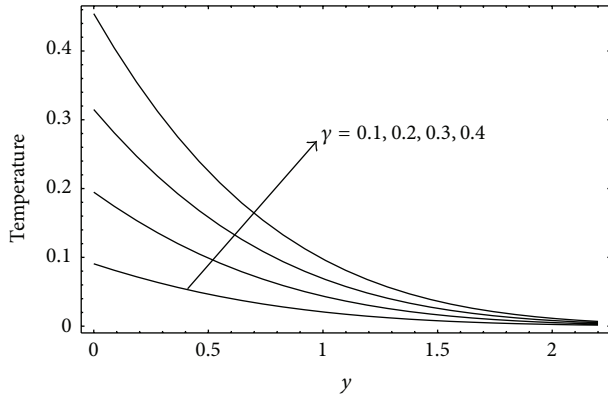


FIGURE 14: Temperature profiles for different values of γ when $t = 0.2$, $R = 1$, and $Pr = 0.71$.

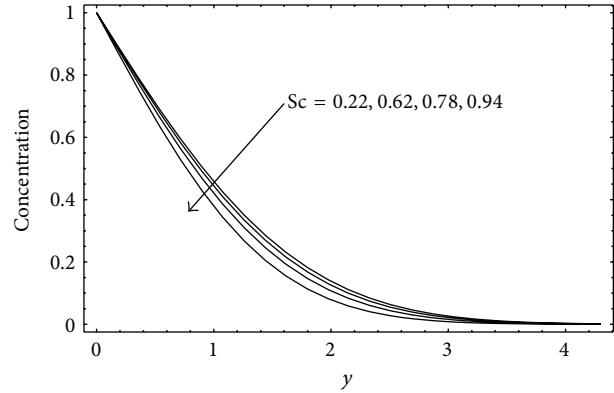


FIGURE 16: Concentration profiles for different values of Sc when $t = 0.2$.

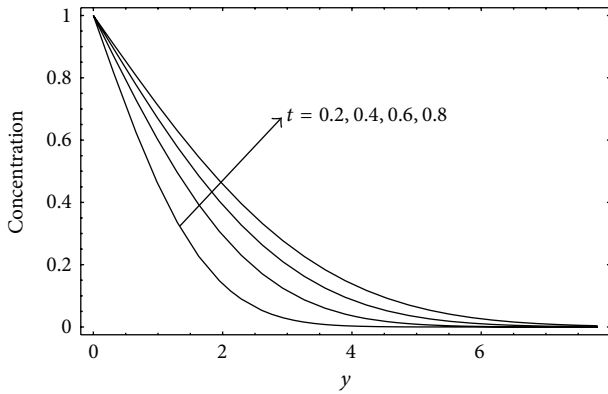


FIGURE 15: Concentration profiles for different values of t when $Sc = 0.22$.

Nomenclature

C' :	Species concentration in the fluid
C_w :	Species concentration near the plate
C_∞ :	Species concentration in the fluid far away from the plate
C :	Dimensionless concentration
c_p :	Heat capacity at a constant pressure
D :	Mass diffusivity
g :	Acceleration due to gravity
h_s :	Heat transfer parameter for Newtonian heating
Gr :	Thermal Grashof number
Gm :	Modified Grashof number
k :	Thermal conductivity of the fluid
Pr :	Prandtl number
q_r :	Radiative heat flux in the y' -direction
R :	Radiation parameter
Sc :	Schmidt number
T' :	Temperature of the fluid
T_∞ :	Ambient temperature
t' :	Time
t :	Dimensionless time
u' :	Velocity of the fluid in the x' -direction
u :	Dimensionless velocity

y' :	Coordinate axis normal to the plate
y :	Dimensionless coordinate axis normal to the plate
k :	Thermal conductivity
k^* :	Mean absorption coefficient
β :	Volumetric coefficient of thermal expansion
β^* :	Volumetric coefficient of mass expansion
μ :	Coefficient of viscosity
ν :	Kinematic viscosity
ρ :	Fluid density
σ :	Stefan-Boltzmann constant
τ :	Skin friction
τ' :	Dimensionless skin friction
θ :	Dimensionless temperature
ω' :	Frequency of oscillation
ωt :	Phase angle
erf c:	Complementary error function.

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